

# EFFECT OF STRAIN HARDENING LAYER ON PITTING FORMATION UNDER ROLLING CONTACT

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## ABSTRACT

The role of the strain-hardened layer playing in pitting formation during rolling contact is analyzed using the fracture mechanics approach. The other governing factors are the initial crack length, crack angle, contact force, friction on all contact surfaces, and the hydraulic pressure of trapped fluid acting on the crack surfaces. The strain energy density factors are calculated by application of the two-dimensional finite element method. The strain energy density theory is applied to show the coupling effects between the hardened layer and the other factors. All analytical results predicted in this paper agree well with the experimental observations.

## 1 INTRODUCTION

The research on rolling contact problem with defects has been a long time and extensive [1-5]. Three layers, ratchetting layer, elastic/plastic shakedown layer and elastic layer, are formed near the surface of substrate. In this paper, the coupling effects of strain-hardened layer with the initial crack length, crack angle, contact force, friction on all contact surfaces, and the hydraulic pressure of trapped fluid applied on the crack surfaces, on pit formation are studied. The strain energy density theory [6] is used to predict the fracture behavior of the surface crack under rolling contact.

## 2 PROBLEM STATEMENT

The two-dimensional model used in this analysis is illustrated in Figure 1. A rigid circular cylindrical roller runs over an elastic plate of  $W = 1.5$  mm long and  $H = 0.55$  mm high represents the elastic half-space. The Young's modulus  $E_p$  and the Poisson's ratio  $\nu_p$  of the elastic plate are 282 GPa and 0.3, respectively. An inclined surface crack of length  $a$  makes an angle  $\beta$  with the surface in rolling  $x$ -direction. The rigid cylindrical roller with radius  $R = 2.49$  mm initially indents on the plate surface with downward displacement  $d$ . The roller is assumed to have a downward displacement  $d$  such that the vertical reaction force between the roller and half surface is  $P = 1957$  N. In this paper,  $d = 0.006$  mm is used for  $P = 1957$  N. After the indentation, the roller runs at a velocity  $V = 0.0838$  mm/sec and rotates at an angular velocity  $\omega = 0.01$  radian/sec from left end position  $x = -0.2$  mm to the right  $x = 0.3028$  mm. The bottom of the plate is assumed fixed and the two sides are allowed to move in  $y$ -direction only. The depth  $h$  denotes the thickness of the strain-hardened layer. In the analysis, the

Poisson's ratio  $\nu_L$  of the layer is assumed to be equal to  $\nu_p$  while the Young's modulus  $E_L$  is subjected to change.

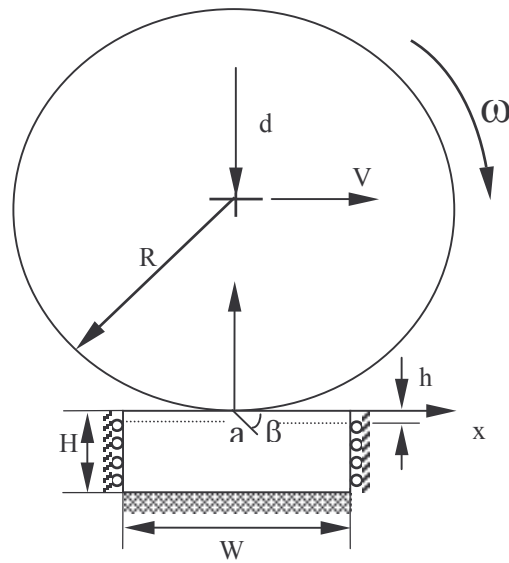


Figure 1. Schematic of rigid circular roller on cracked plate.

## RESULTS AND DISCUSSIONS

### 3.1. Critical pressure

Before going into the discussion of the results, the term "critical pressure" used to characterize the pitting phenomena has to be defined. Figures 2 and 3 show a typical case of the occurrences of  $k_i$  and  $S_{\min}$  vs. location of the rigid body center  $x$  for  $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $d = 0.006$  mm,  $\mu = 0.1$  and  $p = 1020$  MPa. Consider the point A in Figure 2, where the value of  $k_1$  is maximum and at the same time  $k_2$  is negative. If the minimum strain energy density  $S_{\min}$  is computed at that point, it will be shown in Figure 3, i.e. point A. According to strain energy density theory, as  $S_{\min}$  reaches  $S_C$ , the crack will propagate along a positive angle  $\theta_0$ . It means that the crack will propagate toward the horizontal direction. If the analyses are kept on going, the final crack path becomes a pit.

Now, consider the point B in Figure 2 where  $k_2$  is maximum and positive. At this time, the crack is totally compressed and  $k_1$  is almost zero. Its  $S_{\min}$  is also shown in Figure 3. Again, as  $S_{\min}$  reaches  $S_C$ , the crack will propagate downward. In some cases,  $k_1$  increases abruptly after running over the crack mouth and reaches maximum at point C. After point C,  $k_1$  decreases to zero. In this period,  $k_2$  is positive. The strain energy density theory predicts that the crack would propagate downward.

If the hydraulic pressure  $p$  increases, the values of  $k_i$  and  $S_{\min}$  at point A will increase. As  $S_{\min}$  at point A greater than that at point B or C, the pitting will occur and the  $p$  is defined as the "critical pressure"

$p_{cr}$ .

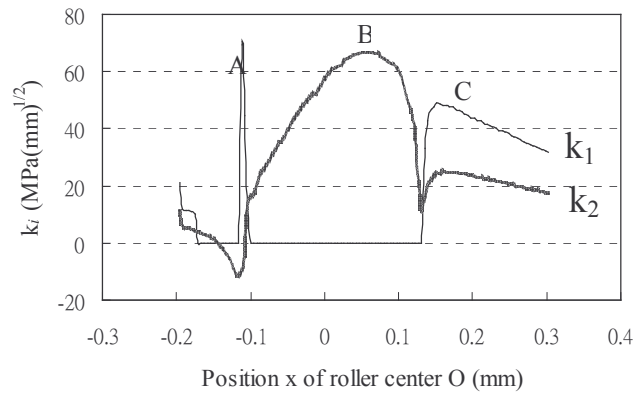


Figure 2. Variations of stress intensity factors  $k_1$  and  $k_2$  with roller center O ( $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $d = 0.006$  mm and  $\mu = 0.1$ ).

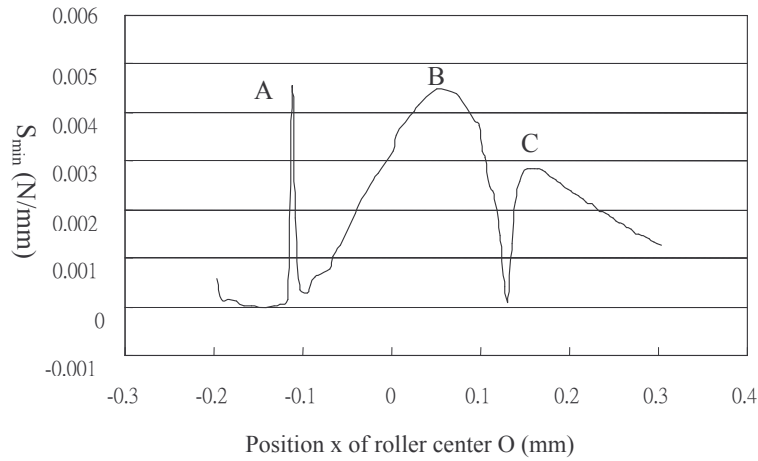


Figure 3. Variations of minimum strain energy density factor  $S_{min}$  with roller center O ( $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $d = 0.006$  mm and  $\mu = 0.1$ ).

### 3.2. Indentation force

For  $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $\mu = 0.05$  and  $h = 0.01$  mm, Figure 4 shows the relation between the critical hydraulic pressure and indentation depth  $d$  when  $E_L = E_p$ ,  $1.4E_p$  and  $3E_p$ . For higher indentation force, it requires larger critical hydraulic pressure to open the crack. Also, as the hardened layer becomes stiffer (i.e. higher  $E_L$ ), it results in smaller crack opening displacement and  $S_{min}$  at point A. For creating a pit, higher critical pressure is needed to make  $(S_{min})_A$  larger than  $(S_{min})_B$  or  $(S_{min})_C$ .

### 3.3. Initial crack length

The effect of initial crack length  $a$  on the formation of pitting can be determined if the hardened layer is considered. It depends on the fact that the crack tip is inside the layer or not. For  $\beta = 25^\circ$ ,  $d = 0.006$

mm,  $\mu = 0.05$  and  $h = 0.01$  mm, Figure 5 shows the relations between the critical pressure and the initial crack length when  $E_L = E_p$ ,  $1.4E_p$  and  $3E_p$ . If  $a = 0.012$  mm, higher  $p_{cr}$  is required for stiffer layer. In the cases of  $a = 0.06$ mm or larger, the crack tip is far away from the interface of hardened layer and base material. It makes no difference to consider the existence of the layer. More complicate situations are arisen when  $a = 0.024$ mm or  $0.036$ mm.

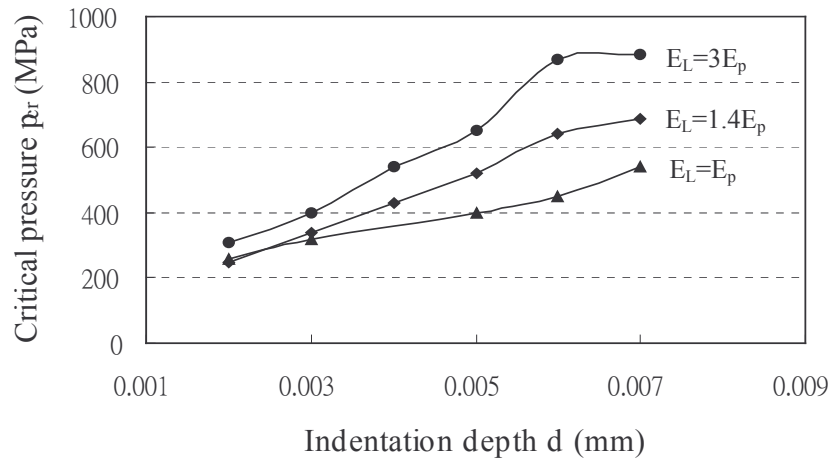


Figure 4. Variations of critical pressure  $p_{cr}$  with indentation depth  $d$  for different  $E_L$  ( $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $\mu = 0.05$  and  $h = 0.01$  mm).

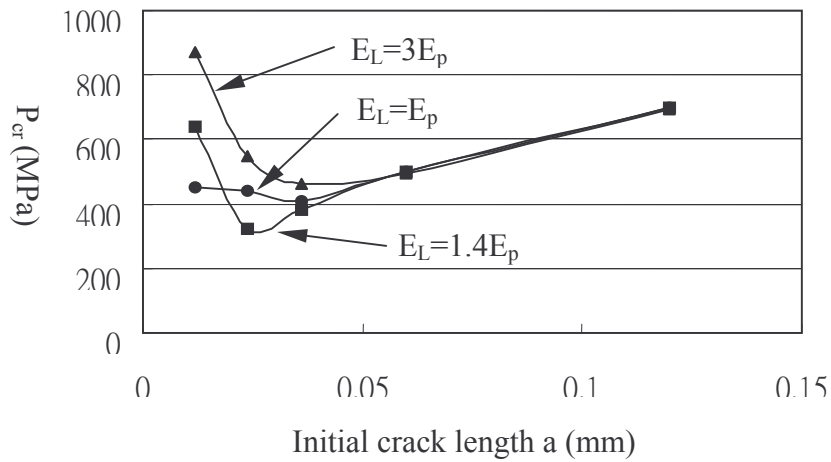


Figure 5. Critical pressure  $p_{cr}$  with crack length  $a$  at different  $E_L$  ( $\beta = 25^\circ$ ,  $d = 0.006$ ,  $\mu = 0.05$  and  $h = 0.01$  mm).

### 3.4. Coefficient of friction

Figure 6 shows the relation between critical pressure and coefficient of friction  $\mu$  at different  $E_L$  for  $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $d = 0.006$  mm and  $h = 0.01$  mm. For dry rollers with high  $\mu$  [5], pit did not form. Pit formation was rapid in good lubricant conditions where friction was low. This conclusion can be seen from Fig. 9 that higher critical pressure is required for higher coefficient of friction to form pits. As the strain hardened surface layer becomes stiffer, it requires higher hydraulic pressure to create a pit. At the condition  $E_L = 3E_p$  and  $\mu = 0.4$  which is not shown in the figure,  $(S_{min})_A$  will never exceed  $(S_{min})_B$ .

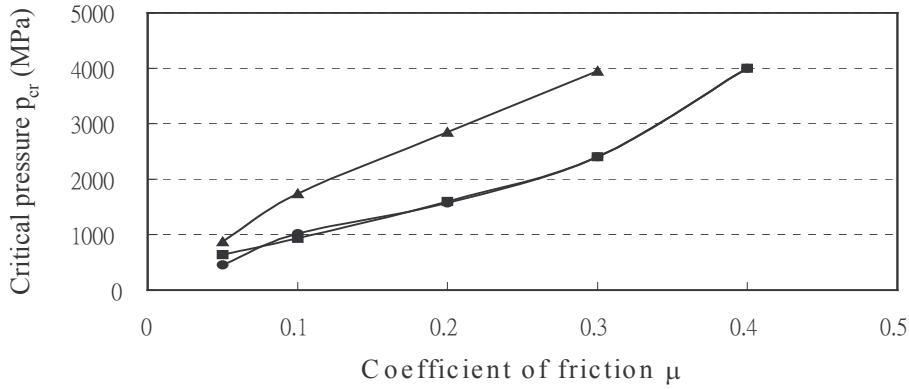


Figure 6. Critical pressure  $p_{cr}$  with  $\mu$  at different  $E_L$  ( $a = 0.012$  mm,  $\beta = 25^\circ$ ,  $d = 0.006$  mm and  $h = 0.01$  mm).

### 3.5. Strain hardened surface layer thickness

Figure 12 plots the variation of  $p_{cr}$  with layer thickness  $h$ . It shows that the critical pressure decreases initially and then increases. For the cases with  $h < 0.1$  mm, the layer behaves as a beam rested on the elastic foundation (the base material). As the roller runs after the crack mouth, the crack opens due to bending effect. At this moment (i.e. point C),  $k_1$  in Figure 3 reaches a maximum and  $(S_{min})_C > (S_{min})_B$ . Consequently, the hydraulic pressure has to be high enough such that  $(S_{min})_A > (S_{min})_C$ . From the finite element results, it shows that  $(S_{min})_C$  decreases with increasing thickness  $h$  ( $h < 0.1$  mm). As a result, the required hydraulic pressure for creating a pit becomes small when  $h < 0.1$  mm.

As  $h \geq 0.1$  mm, most of the plate becomes stiffer. The intensity  $k_2$  at point B in Figure 3 dominates and  $(S_{min})_B > (S_{min})_C$ . Consequently, the pressure acting on the crack surface should be large enough such that  $(S_{min})_A > (S_{min})_B$ . Thus, high critical pressure is required for  $h \geq 0.1$  mm.

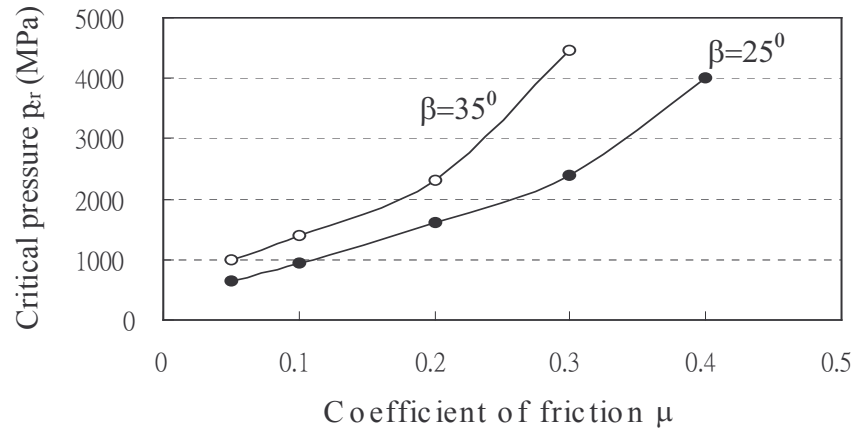


Figure 11. Variations of critical pressure  $p_{cr}$  with coefficient of friction  $\mu$  for different crack angles ( $a = 0.012$  mm,  $d = 0.006$  mm,  $h = 0.01$  mm and  $E_L = 1.4E_p$ ).

#### 5 REFERENCES

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